



# Digital Control

CSE421

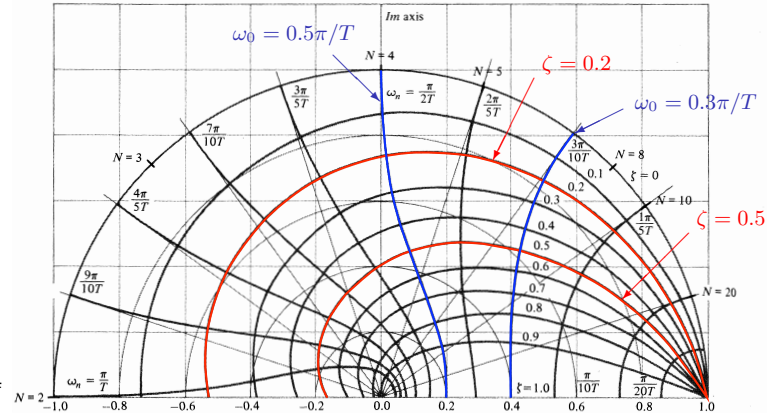
Assoc. Prof. Dr.Ing.

**Mohammed Ahmed**

mnahmed@eng.zu.edu.eg

goo.gl/GHZZio

## Lecture 5: Step response and pole locations

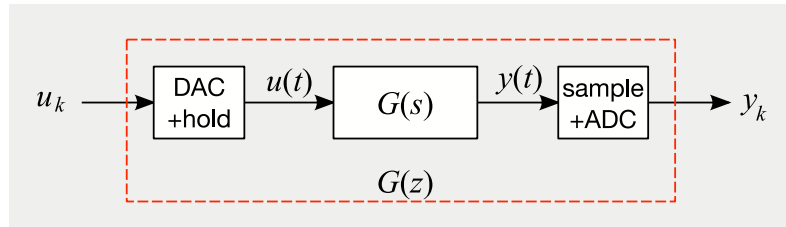


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# Review

- Definition of **z-transform**:  $U(z) = \mathcal{Z}\{u_k\} = \sum_{k=0}^{\infty} u_k z^{-k}$
- Discrete transfer function:  $\frac{Y(z)}{U(z)} = G(z) = \mathcal{Z}\{g_k\}$ ,  $g_k =$  **pulse response**
- Construct a discrete model of a continuous sampled-data system  $G(s) \dots$



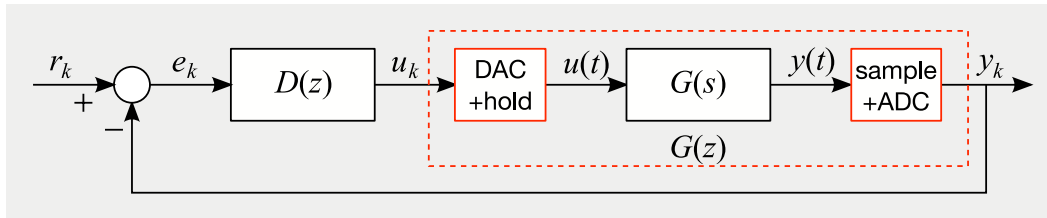
... by computing the pulse response  $g_k$  and transforming to get  $G(z)$ :

$$G(z) = (1 - z^{-1}) \mathcal{Z} \left\{ \frac{G(s)}{s} \right\}$$

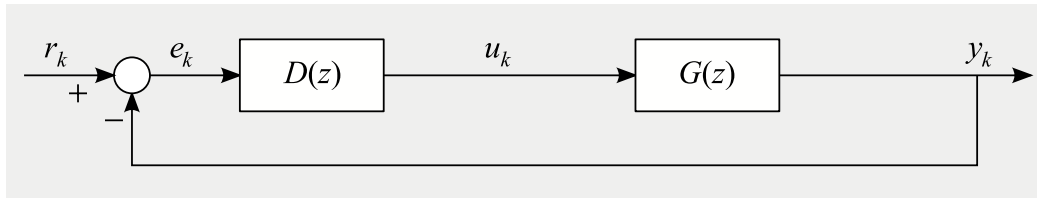
- Output response:  $Y(z) = G(z)U(z) \iff y_k = g_k * u_k$

# Review

Analyse/design a discrete controller  $D(z)$ :



by considering the purely discrete time system:



Closed loop system transfer function: 
$$\frac{Y(z)}{R(z)} = \frac{G(z)D(z)}{1 + G(z)D(z)}$$

How do the closed loop poles relate to

- stability?
- performance?

# Response of 2nd order system

Consider the z-transform of a sinusoid multiplied by an exponential signal:

$$y(t) = e^{-at} \cos(bt) \mathcal{U}(t) \quad (\mathcal{U}(t) = \text{unit step})$$

★ sample:  $y(kT) = r^k \cos(k\theta) \mathcal{U}(kT)$  with  $r = e^{-aT}$  &  $\theta = bT$

★ transform: 
$$Y(z) = \frac{1}{2} \frac{z}{(z - re^{j\theta})} + \frac{1}{2} \frac{z}{(z - re^{-j\theta})}$$

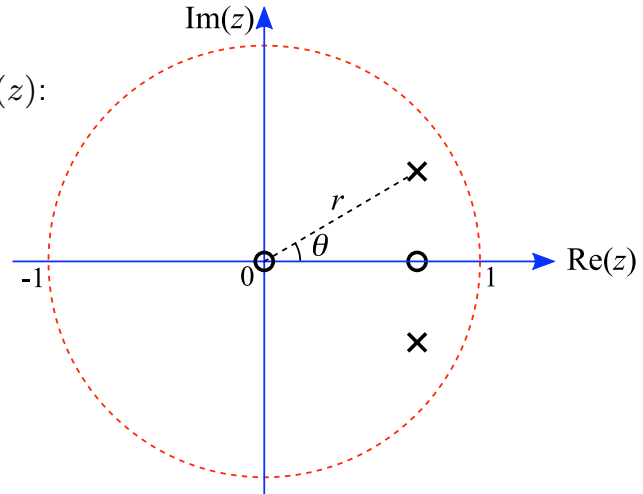
$$= \frac{z(z - r \cos \theta)}{(z - re^{j\theta})(z - re^{-j\theta})}$$

★ e.g.  $y_k$  is the pulse response of  $G(z)$ :

$$G(z) = \frac{z(z - r \cos \theta)}{(z - re^{j\theta})(z - re^{-j\theta})}$$

poles:  $\begin{cases} z = re^{j\theta} \\ z = re^{-j\theta} \end{cases}$

zeros:  $\begin{cases} z = 0 \\ z = r \cos \theta \end{cases}$



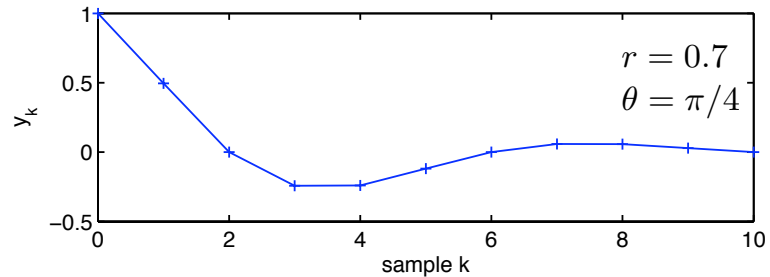
# Response of 2nd order system

Responses for varying  $r$ :

▷  $r < 1$



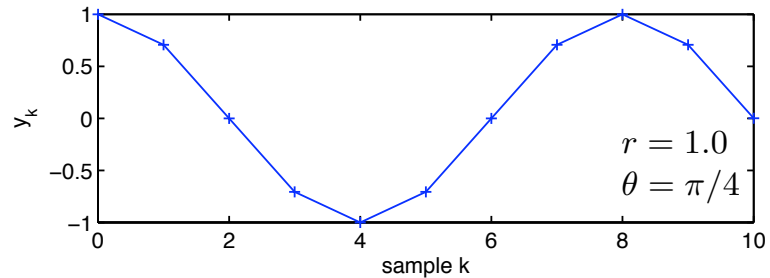
exponentially decaying envelope



▷  $r = 1$



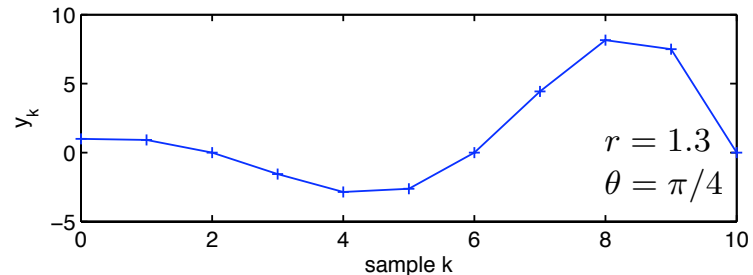
sinusoidal response with  $2\pi/\theta$  samples per period



▷  $r > 1$



exponentially increasing envelope



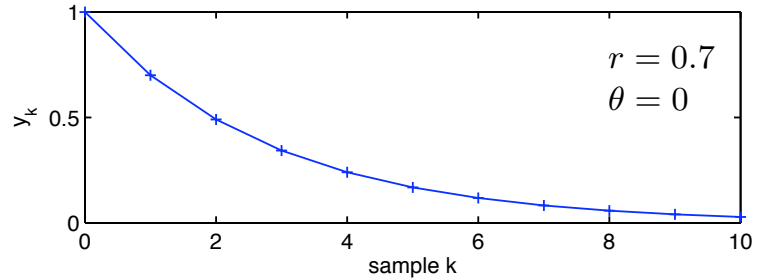
# Response of 2nd order system

Responses for varying  $\theta$ :

▷  $\theta = 0$



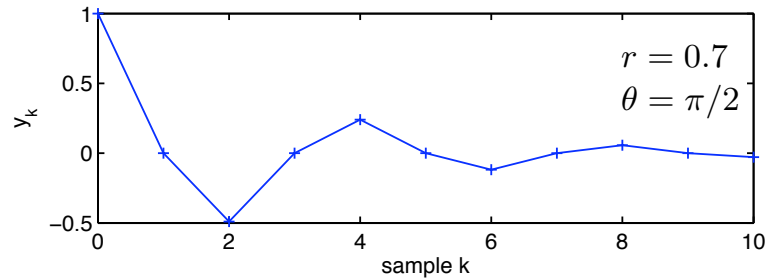
decaying exponential



▷  $\theta = \pi/2$



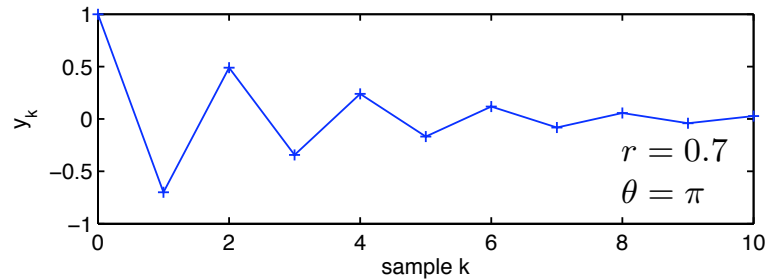
$2\pi/\theta = 4$  samples  
per period



▷  $\theta = \pi$



2 samples per period



## Response of 2nd order system

Some special cases:

- ▷ for  $\theta = 0$ ,  $Y(z)$  simplifies to:

$$Y(z) = \frac{z}{z - r}$$

⇒ exponentially decaying response

- ▷ when  $\theta = 0$  and  $r = 1$ :

$$Y(z) = \frac{z}{z - 1}$$

⇒ unit step

- ▷ when  $r = 0$ :

$$Y(z) = 1$$

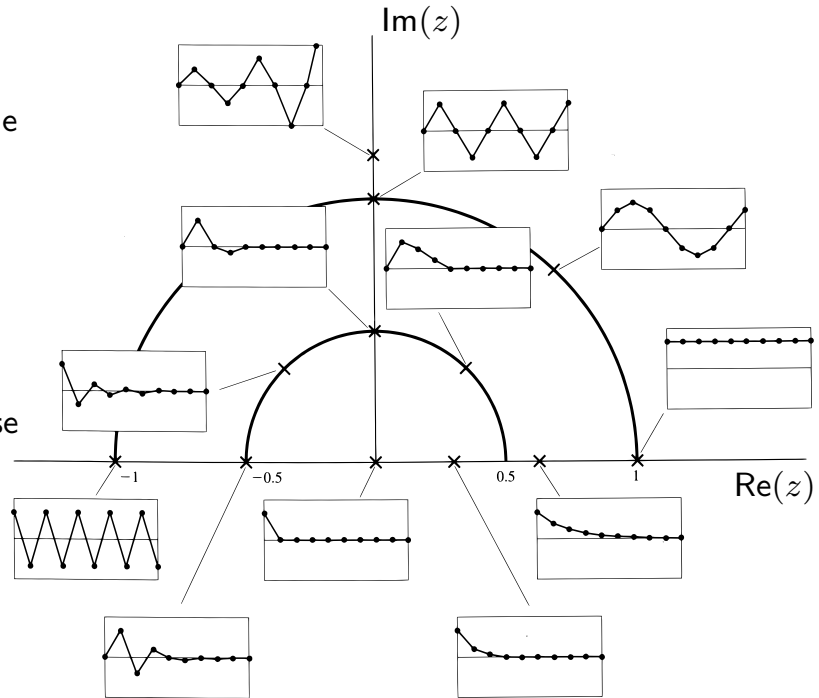
⇒ unit pulse

- ▷ when  $\theta = 0$  and  $-1 < r < 0$ :

samples of alternating signs

# Pole positions in the z-plane

- Poles inside the unit circle are **stable**
- Poles outside the unit circle are **unstable**
- Poles on the unit circle are **oscillatory**
- Real poles at  $0 < z < 1$  give exponential response
- Higher frequency of oscillation for larger  $\theta$
- Lower apparent damping for larger  $\theta$  and  $r$





# Relationship with s-plane poles

If  $F(s)$  has a pole at  $s = a$   
 then  $F(z)$  has a pole at  $z = e^{aT}$

↑  
 consistent with  $z = e^{sT}$

What about transfer functions?

$$G(z) = (1 - z^{-1}) \mathcal{Z} \left\{ \frac{G(s)}{s} \right\}$$

↓

If  $G(s)$  has poles  $s = a_i$   
 then  $G(z)$  has poles  $z = e^{a_i T}$

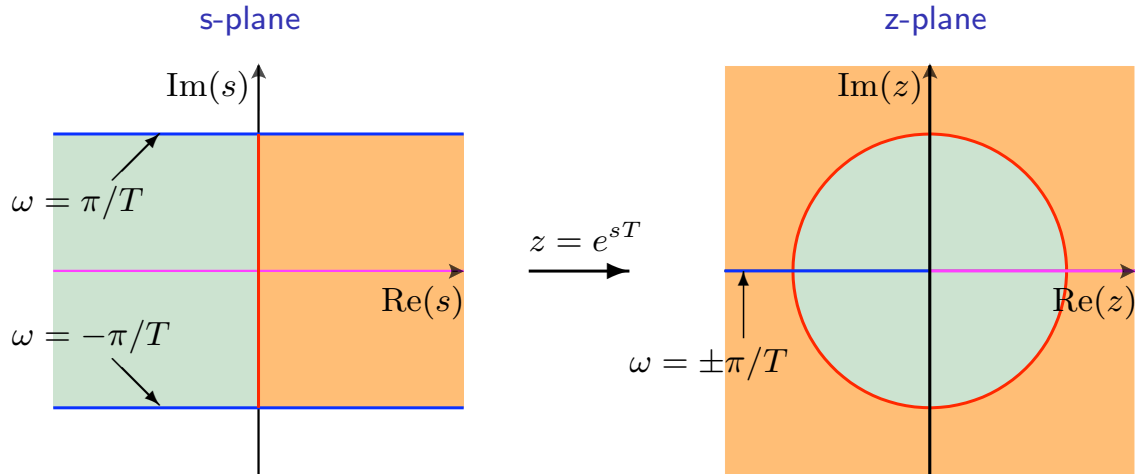
but the zeros are unrelated

$\mathcal{F}(s)$	$f(kT)$	$F(z)$
$\frac{1}{s}$	$1(kT)$	$\frac{z}{z-1}$
$\frac{1}{s^2}$	$kT$	$\frac{Tz}{(z-1)^2}$
$\frac{1}{s+a}$	$e^{-akT}$	$\frac{z}{z-e^{-aT}}$
$\frac{1}{(s+a)^2}$	$kT e^{-akT}$	$\frac{Tz e^{-aT}}{(z-e^{-aT})^2}$
$\frac{a}{s(s+a)}$	$1 - e^{-akT}$	$\frac{z(1 - e^{-aT})}{(z-1)(z-e^{-aT})}$
$\frac{b-1}{(s+a)(s+b)}$	$e^{-akT} - e^{-bkT}$	$\frac{(e^{-aT} - e^{-bT})z}{(z-e^{-aT})(z-e^{-bT})}$
$\frac{a}{s^2 + a^2}$	$\sin akT$	$\frac{z \sin aT}{z^2 - (2 \cos aT)z + 1}$
$\frac{b}{(s+a)^2 + b^2}$	$e^{-akT} \sin bkT$	$\frac{z e^{-aT} \sin bT}{z^2 - 2e^{-aT}(\cos bT)z + e^{-2aT}}$

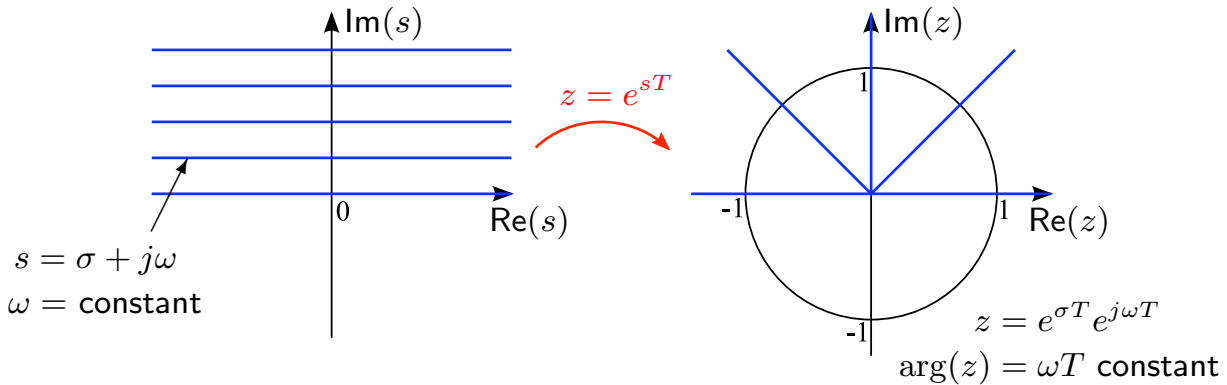
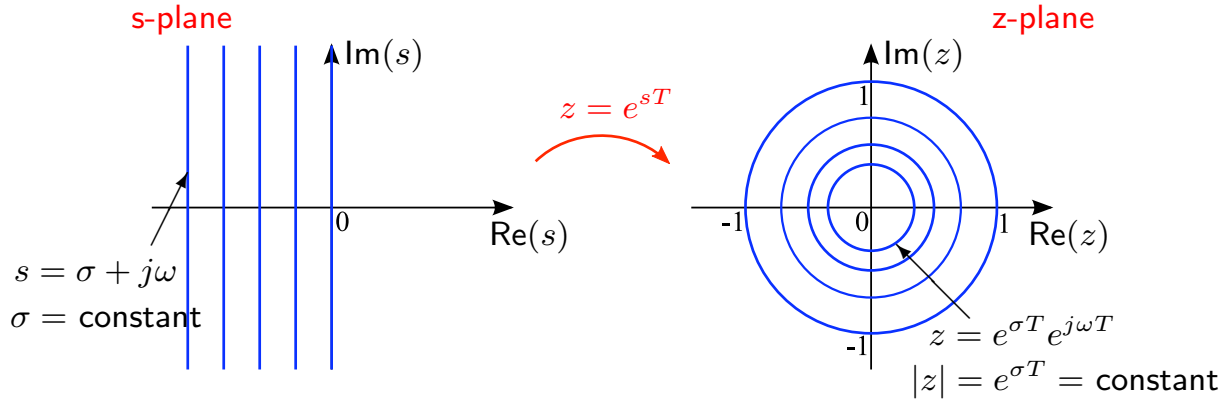
# The mapping from s-plane to z-plane

Locus of  $s = \sigma + j\omega$  under the mapping  $z = e^{sT}$ :

- ★ imaginary axis ( $s = j\omega, \sigma = 0$ )  $\rightarrow$  unit circle ( $|z| = 1$ )
- ★ **left-half** plane ( $\sigma < 0$ )  $\rightarrow$  **inside** of unit circle ( $|z| < 1$ )
- ★ **right-half** plane ( $\sigma > 0$ )  $\rightarrow$  **outside** of unit circle ( $|z| > 1$ )
- ★ region of s-plane within the Nyquist rate ( $|\omega| < \pi/T$ )  $\rightarrow$  entire z-plane



# The mapping from s-plane to z-plane



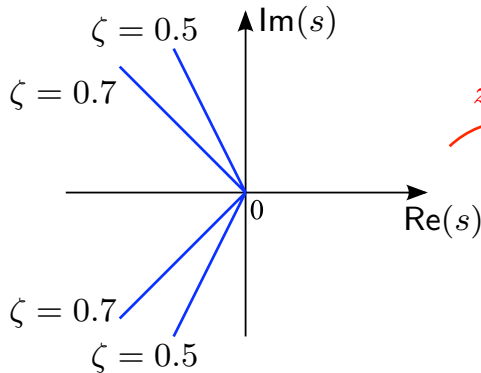
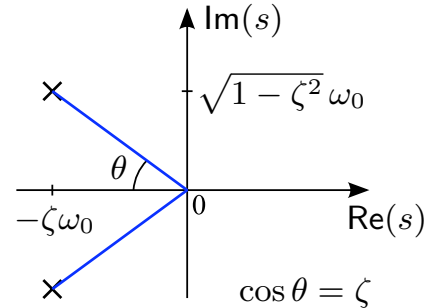
# The mapping from s-plane to z-plane

Pole locations for constant damping ratio  $\zeta < 1$

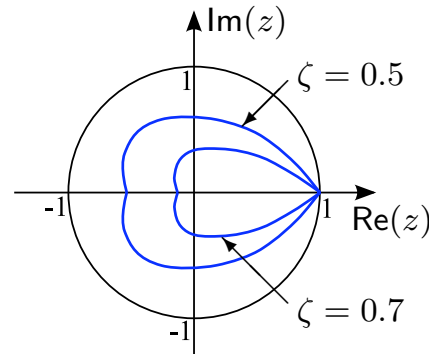
$$s^2 + \zeta\omega_0 s + \omega_0^2 = 0$$

$\Downarrow$

$$s = -\zeta\omega_0 \pm j\sqrt{1 - \zeta^2}\omega_0$$



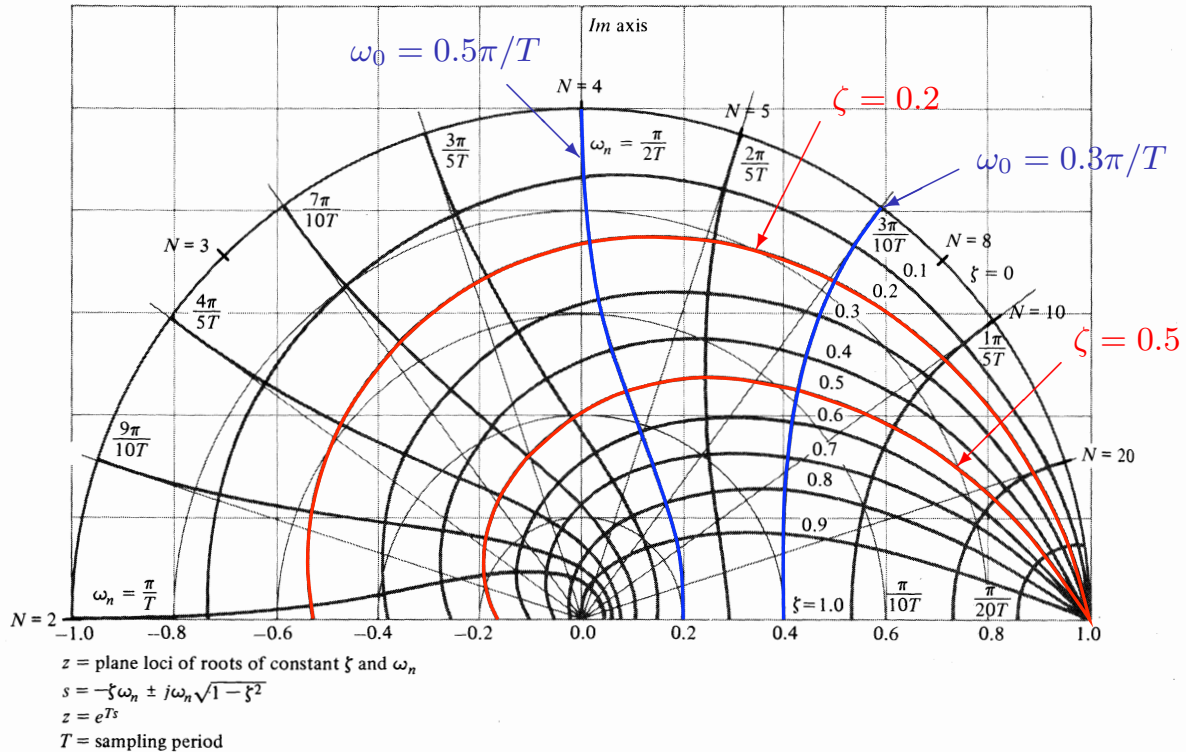
$z = e^{sT}$



$$s = -\zeta\omega_0 + j\sqrt{1 - \zeta^2}\omega_0: \zeta = \text{constant}$$

$$z = e^{-\zeta\omega_0 T} e^{-j\sqrt{1 - \zeta^2}\omega_0 T}$$

# The mapping from s-plane to z-plane

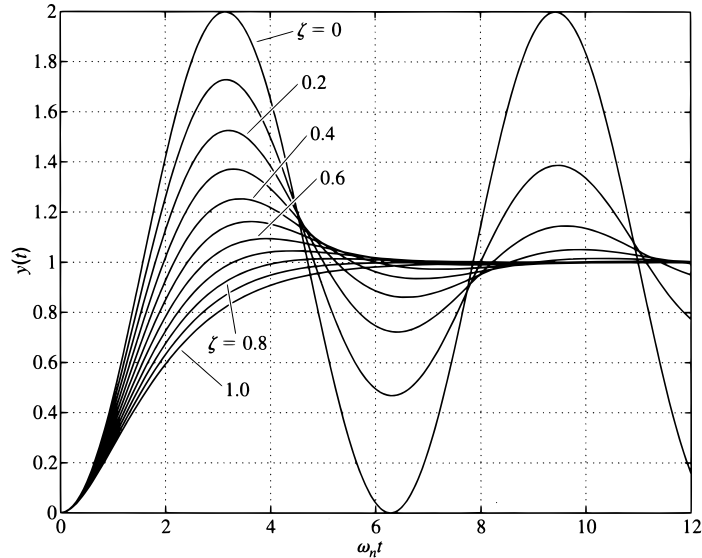


# The mapping from s-plane to z-plane



# System specifications

Second order step responses (e.g. see HLT)

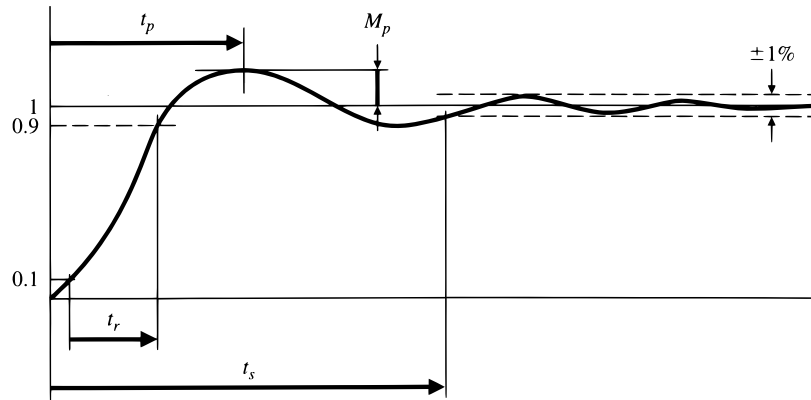


Design criteria based on step response:

- ★ Damping ratio  $\zeta$  in range 0.5 – 0.9 [application-dependent]
- ★ Natural frequency  $\omega_0$  as large as possible [for fastest response]

# System specifications

Typical specifications for the step response:

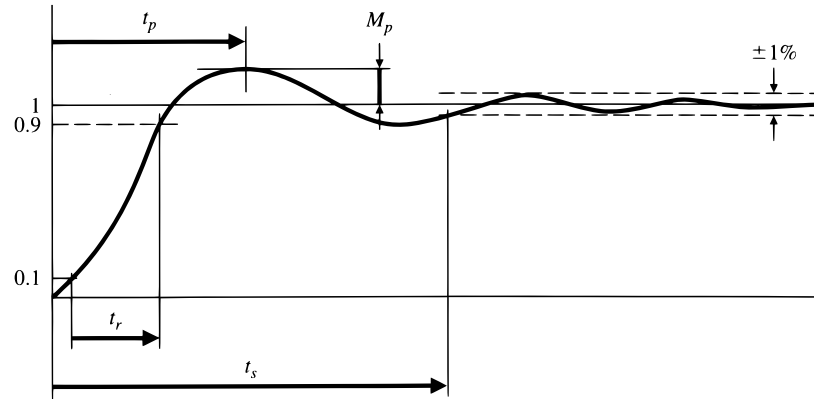


- ★ Rise time (10% → 90%):  $t_r \approx 1.8/\omega_0$
- ★ Peak overshoot:  $M_p \approx e^{-\pi\zeta/\sqrt{1-\zeta^2}}$
- ★ Settling time (to 1%):  $t_s = 4.6/(\zeta\omega_0)$
- ★ Steady state error to unit step:  $e_{ss}$
- ★ Phase margin:  $\phi_{PM} \approx 100\zeta$



# System specifications

Typical specifications for the step response:



$t_r, M_p \rightarrow \zeta, \omega_0 \rightarrow$  locations of dominant poles

$t_s \rightarrow$  radius of poles:  $|z| < 0.01^{T/t_s}$

$e_{ss} \rightarrow$  final value theorem:  $e_{ss} = \lim_{z \rightarrow 1} (z - 1)E(z)$

# System specifications

**Example** – A continuous system with transfer function

$$G(s) = \frac{1}{s(10s + 1)}$$

is controlled by a discrete control system with a ZOH

The closed loop system is required to have:

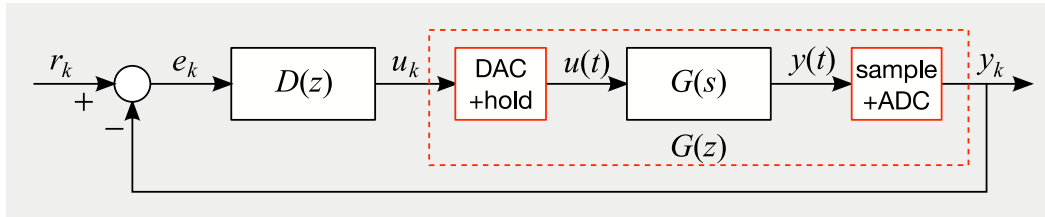
- step response overshoot:  $M_p < 16\%$
- step response settling time (1%):  $t_s < 10$  s
- steady state error to unit ramp:  $e_{ss} < 1$

Check these specifications if  $T = 1$  s and the controller is

$$u_k = -0.5u_{k-1} + 13(e_k - 0.88e_{k-1})$$

# System specifications

1. (a) Find the pulse transfer function of  $G(s)$  plus the ZOH



$$G(z) = (1 - z^{-1}) \mathcal{Z} \left\{ \frac{G(s)}{s} \right\} = \frac{(z - 1)}{z} \mathcal{Z} \left\{ \frac{0.1}{s^2(s + 0.1)} \right\}$$

e.g. look up  $\mathcal{Z}\{a/s^2(s + a)\}$  in tables:

$$\begin{aligned} G(z) &= \frac{(z - 1)}{z} \frac{z \left( (0.1 - 1 + e^{-0.1})z + (1 - e^{-0.1} - 0.1e^{-0.1}) \right)}{0.1(z - 1)^2(z - e^{-0.1})} \\ &= \frac{0.0484(z + 0.9672)}{(z - 1)(z - 0.9048)} \end{aligned}$$

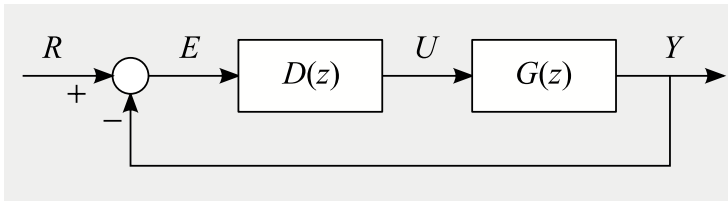
- (b) Find the controller transfer function (using  $z = \text{shift operator}$ ):

$$\frac{U(z)}{E(z)} = D(z) = 13 \frac{(1 - 0.88z^{-1})}{(1 + 0.5z^{-1})} = 13 \frac{(z - 0.88)}{(z + 0.5)}$$

# System specifications

2. Check the steady state error  $e_{ss}$  when  $r_k = \text{unit ramp}$

$$e_{ss} = \lim_{k \rightarrow \infty} e_k = \lim_{z \rightarrow 1} (z - 1)E(z)$$

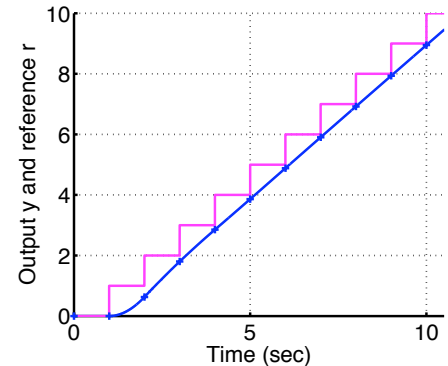


$$\frac{E(z)}{R(z)} = \frac{1}{1 + D(z)G(z)}$$

$$R(z) = \frac{Tz}{(z - 1)^2}$$

$$\begin{aligned} \text{so } e_{ss} &= \lim_{z \rightarrow 1} \left\{ (z - 1) \frac{Tz}{(z - 1)^2} \frac{1}{1 + D(z)G(z)} \right\} = \lim_{z \rightarrow 1} \frac{T}{(z - 1)D(z)G(z)} \\ &= \lim_{z \rightarrow 1} \frac{T}{(z - 1) \frac{0.0484(z + 0.9672)}{(z - 1)(z - 0.9048)} D(1)} \\ &= \frac{1 - 0.9048}{0.0484(1 + 0.9672)D(1)} = 0.96 \end{aligned}$$

$$\Rightarrow e_{ss} < 1 \quad (\text{as required})$$



# System specifications

3. Step response: overshoot  $M_p < 16\% \implies \zeta > 0.5$   
settling time  $t_s < 10 \implies |z| < 0.01^{1/10} = 0.63$

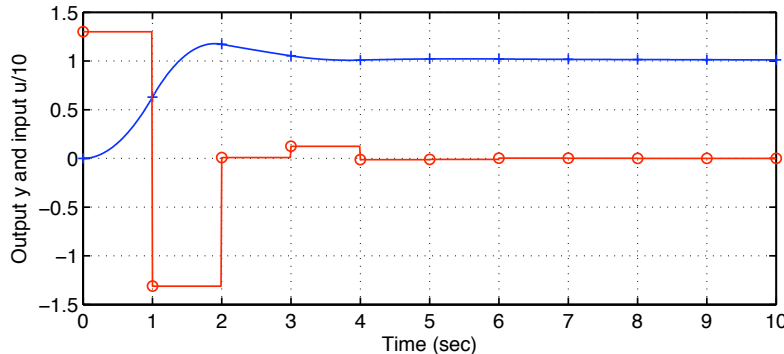
The closed loop poles are the roots of  $1 + D(z)G(z) = 0$ , i.e.

$$1 + 13 \frac{(z - 0.88)}{(z + 0.5)} \frac{0.0484(z + 0.9672)}{(z - 1)(z - 0.9048)} = 0$$

$$\implies z = 0.88, -0.050 \pm j0.304$$

But the pole at  $z = 0.88$  is cancelled by controller zero at  $z = 0.88$ , and

$$z = -0.050 \pm j0.304 = r e^{\pm j\theta} \implies \begin{cases} r = 0.31, \theta = 1.73 \\ \zeta = 0.56 \end{cases}$$



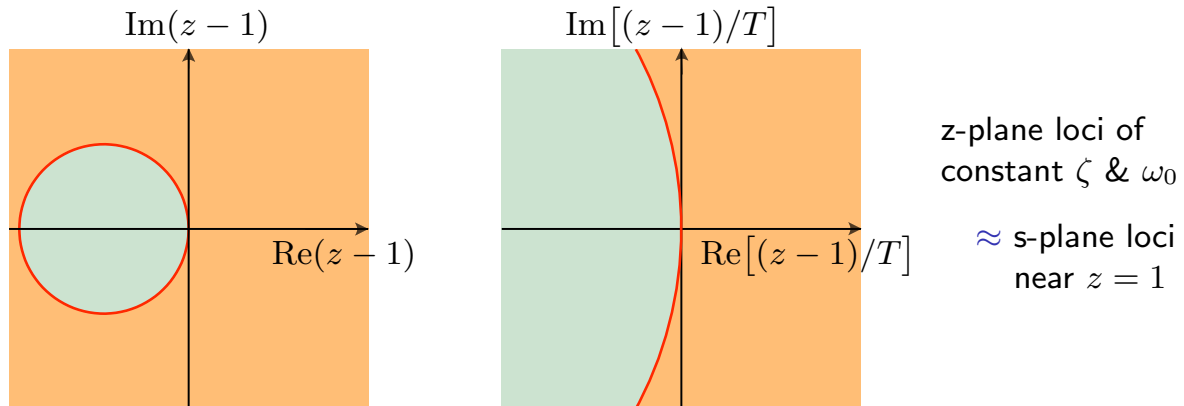
all specs satisfied!

# Fast sampling revisited

For small  $T$ :

$$z = e^{sT} = 1 + sT + (sT)^2/2 + \dots \approx 1 + sT \quad \implies \quad s \approx \frac{z - 1}{T}$$

Hence the image of the unit circle under the map from  $z$  to  $s$ -plane becomes



but the dominant poles lie near  $z = 1 \dots$

$\dots$  so the discrete response tends to the continuous response as  $T \rightarrow 0$

# Summary

- Dependence of system pulse response on pole locations

- For a sampled data system with a ZOH:

if  $s = a_i$  is a pole of  $G(s)$ , then  $z = e^{a_i T}$  is a pole of  $G(z)$

- Locus of  $s = \sigma + j\omega$  under the mapping  $z = e^{sT}$ :

- ★ the left half plane ( $\sigma < 0$ ) maps to the unit disk ( $|z| < 1$ )

- ★ s-plane poles with damping ratio  $\zeta$ , natural frequency  $\omega_0$  map to z-plane poles with:

$$|z| = e^{-\zeta\omega_0 T}$$

$$\arg(z) = \sqrt{1 - \zeta^2} \omega_0 T$$

- Design specifications (rise time, settling time, overshoot)  
imply constraints on locations of dominant poles